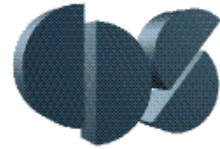




# CDS 101/110: Lecture 10-2

## Loop Shaping Design Example



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2 December 2015

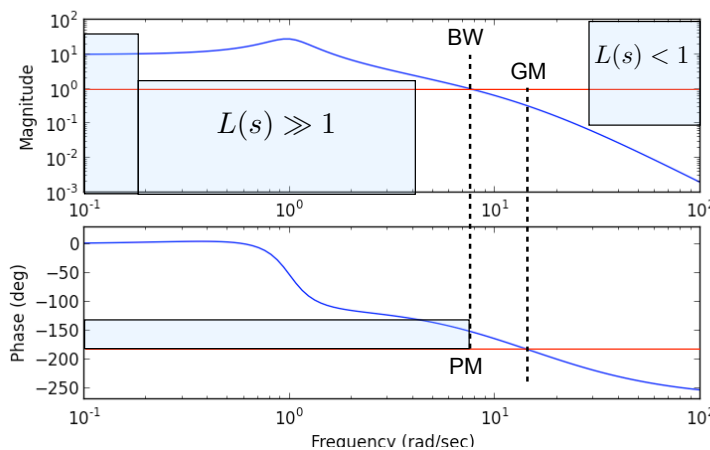
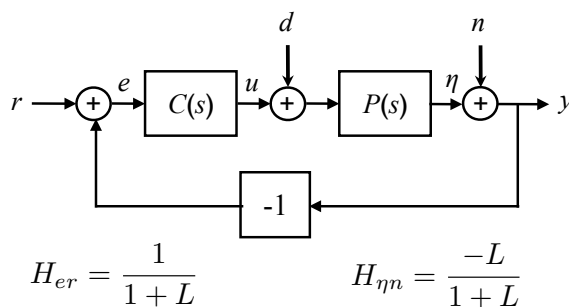
### Goals:

- Work through detailed loop shaping-based design

### Reading:

- Åström and Murray, Feedback Systems, Sec 12.6

## “Loop Shaping”: Design Loop Transfer Function



### Translate specs to “loop shape”

$$L(s) = P(s)C(s)$$

- Design  $C(s)$  to obey constraints

### Typical loop constraints

- High gain at low frequency
  - Good tracking, disturbance rejection at low freqs
- Low gain at high frequency
  - Avoid amplifying noise
- Sufficiently high bandwidth
  - Good rise/settling time
- Shallow slope at crossover
  - Sufficient phase margin for robustness, low overshoot

### Key constraint: slope of gain curve determines phase curve

- Can't independently adjust
- Eg: slope at crossover sets PM

## Example: Control of Vectored Thrust Aircraft



### System description

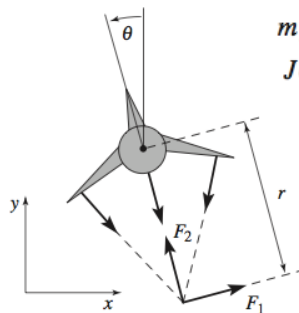
- Vector thrust engine attached to wing
- Inputs: fan thrust, thrust angle (vectored)
- Outputs: position and orientation
- States:  $x, y, \theta$  + derivatives
- Dynamics: flight aerodynamics

### Control approach

- Design “inner loop” control law to regulate pitch ( $\theta$ ) using thrust vectoring
- Second “outer loop” controller regulates the position and altitude by commanding the pitch and thrust
- Basically the same approach as aircraft control laws

## Controller structure

### Full system dynamics

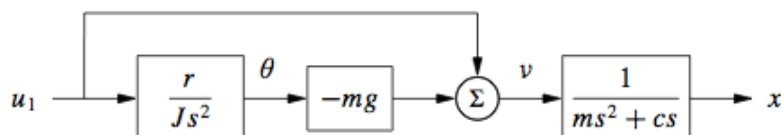


$$\begin{aligned} m\ddot{x} &= -mg \sin \theta - c\dot{x} + u_1 \cos \theta - u_2 \sin \theta, \\ m\ddot{y} &= mg(\cos \theta - 1) - c\dot{y} + u_1 \sin \theta + u_2 \cos \theta, \\ J\ddot{\theta} &= ru_1. \end{aligned}$$

Symbol	Description	Value
$m$	Vehicle mass	4.0 kg
$J$	Vehicle inertia, $\phi_3$ axis	0.0475 kg m <sup>2</sup>
$r$	Force moment arm	25.0 cm
$c$	Damping coefficient	0.05 kg m/s
$g$	Gravitational constant	9.8 m/s <sup>2</sup>

### Simplified lateral dynamics ( $x, \theta$ )

- Linearize the system around hover (equilibrium point)
- Focus on the sideways motion (coupled to roll angle)
- Linearized process dynamics become:



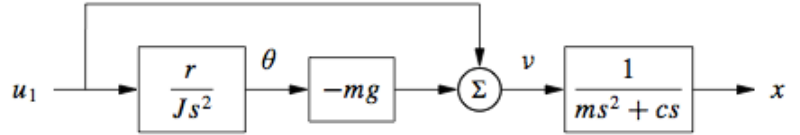
$$H_{\theta u_1} = \frac{r}{Js^2}$$

$$H_{xu_1} = \frac{Js^2 - mgr}{Js^2(ms^2 + cs)}$$

## Control Strategy: Inner/Outer Loop Design

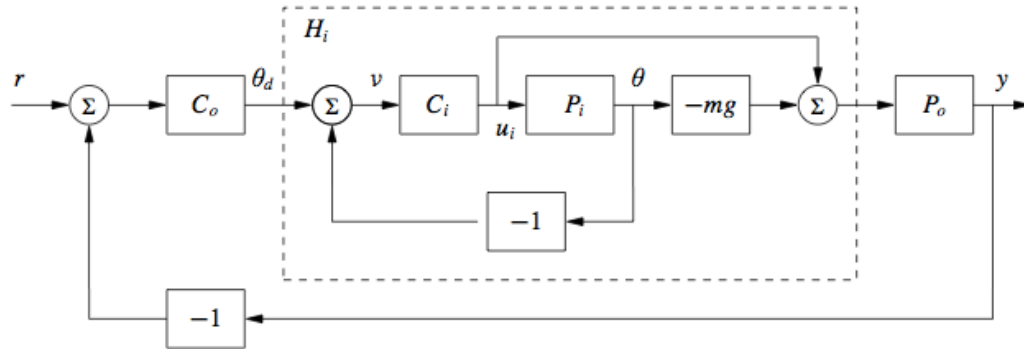
### Control position via roll

- Use “inner” loop to command  $u_1$  so that  $\theta$  tracks a desired value
- Use “outer” loop to command  $\theta$  so that  $x$  tracks a desired value
- Motivation: split the design problem into two simpler pieces



$$H_{xu_1} = \frac{Js^2 - mgr}{Js^2(ms^2 + cs)}$$

### Controller architecture



- Inner loop: design  $C_i$  so that roll angle ( $\theta$ ) tracks  $\theta_d$
- Outer loop: assume roll angle controller is perfect ( $H_i = 1$ ) and then design  $C_o$
- Combine inner and outer loop designs to get overall control system design

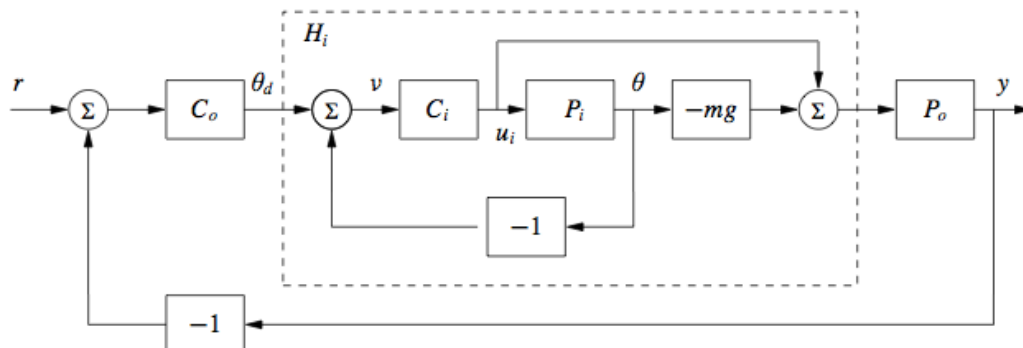
## Controller Specification

### Overall specification (outer loop)

- Zero steady state error for lateral step response
- Bandwidth of approximately 1 rad/sec
- Phase margin of 45 deg (~20% overshoot)

### Inner loop specification: fast tracking of $\theta_d$ (so that outer loop can ignore this)

- Set bandwidth to approximately 10X outer loop = 10 rad/sec
- Low frequency error no more than 5%
- Low overshoot (60 deg phase margin should be enough)



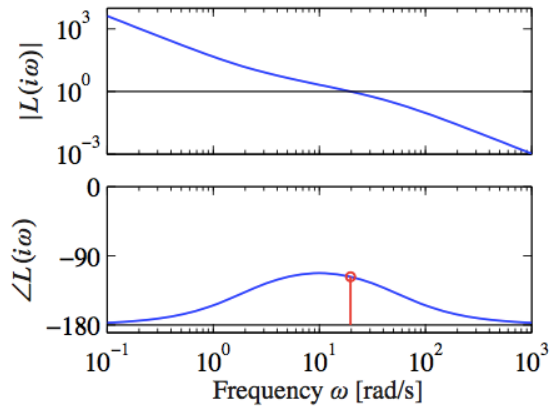
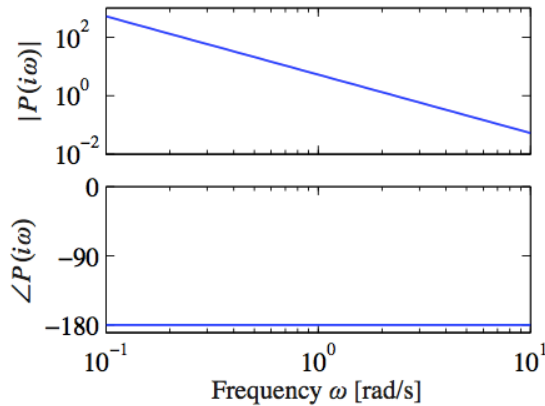
## Inner Loop Design

**Loop shaping: bandwidth > 10 rad/sec, phase margin > 60 deg**

- Process dynamics are second order integrator
- Use lead compensator to add phase;  $a = 2$ ,  $b = 50$ ,  $K = 300$
- Get BW = 20 rad/sec, PM = 60 deg

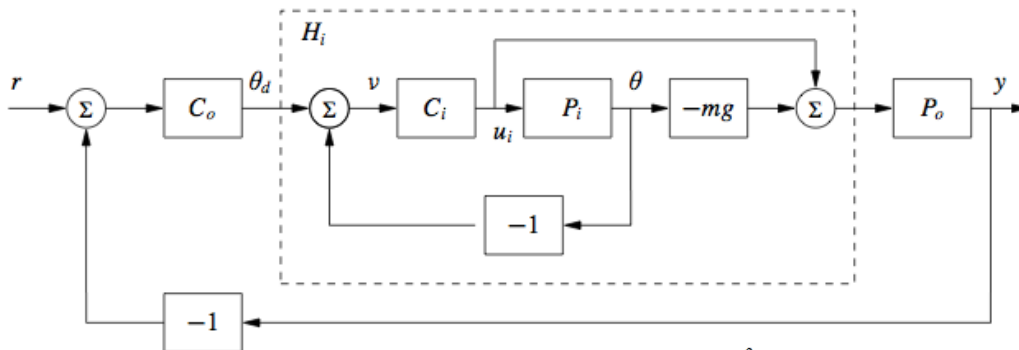
$$H_{\theta u_1} = \frac{r}{Js^2}$$

$$C(s) = K \frac{s+a}{s+b}$$

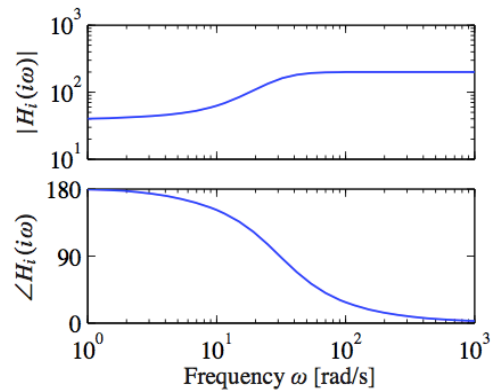
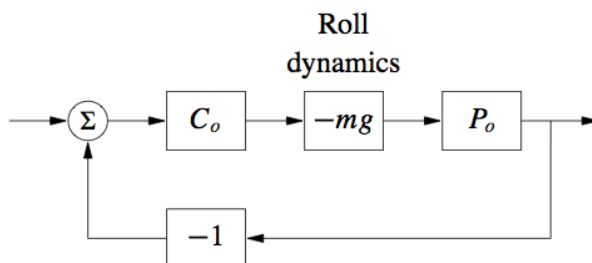


## Simplified Inner Loop Dynamics

**Full dynamics:**

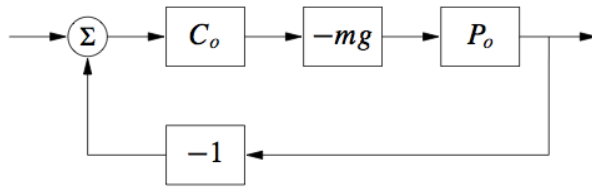


**Reduced dynamics:**



## Outer Loop Design

### Design using simplified dynamics



$$P(s) = H_i(0)P_o(s) = \frac{H_i(0)}{ms^2 + cs}$$

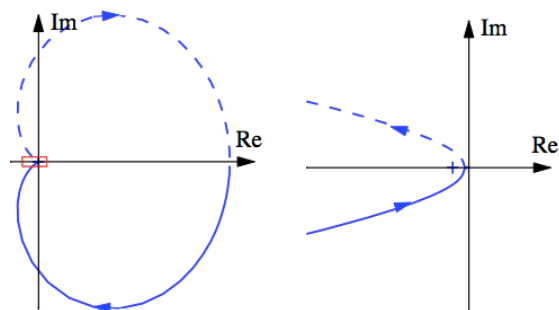
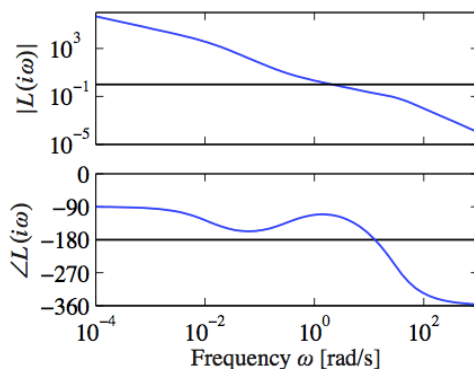
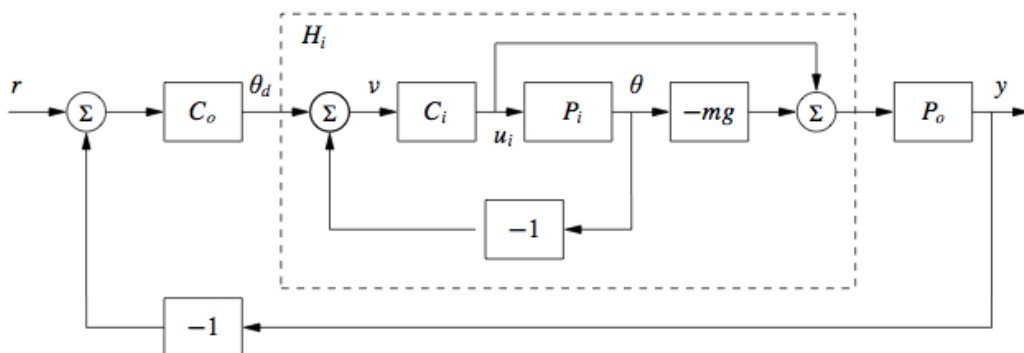
$$C_o(s) = -k_o \frac{s + a_o}{s + b_o}$$

- Process dynamics are (approximately) a double integrator (again!)
- Control design specs
  - Zero steady state error for lateral step response
  - Bandwidth of approximately 1 rad/sec
  - Phase margin of 45 deg (~20% overshoot)
- Can use a lead compensator (again!): put phase lead around 1 rad/sec
- $a_o = 0.3$ ,  $b_o = 10$ ,  $K_l = 2 \Rightarrow$  get > 60 deg phase margin, with BW  $\approx 1$

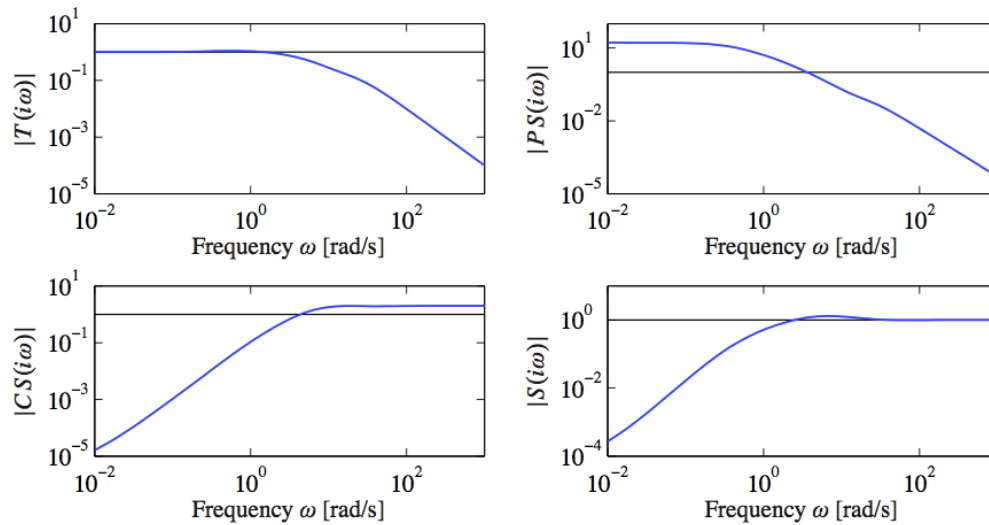
### Remarks

- Note that we will have some residual phase lag from  $H_i(s)$  at  $\omega = 1 \Rightarrow$  set PM = 60 to give a bit of additional margin
- Need to check that the design works with  $H_o(s)$  replaced by  $H_i(s)$

## Final System Design



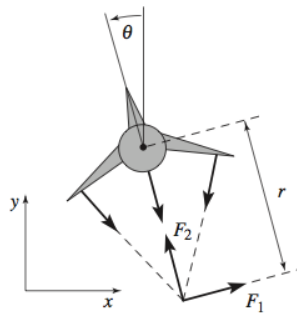
## Final Check: Gang of 4



### Remarks

- $|PS|$  is a bit large at low frequency  $\Rightarrow$  poor disturbance rejection
  - At low frequency  $C(s) = \text{constant} \Rightarrow P / (1 + PC) \rightarrow 1/C$
- Can fix this by using integral compensation in outer loop controller

## Summary



### Overall specification (outer loop)

- Zero steady state error for lateral step response
- Bandwidth of approximately 1 rad/sec
- Phase margin of 45 deg (~20% overshoot)

$$H_{xu_1} = \frac{Js^2 - mgr}{Js^2(ms^2 + cs)}$$

